NATIONAL ADVISORY COMMITTEE FOR AERONAUTICS

TECHNICAL NOTE 2389

FATIGUE STRENGTHS OF AIRCRAFT MATERIALS

AXIAL-LOAD FATIGUE TESTS ON NOTCHED SHEET SPECIMENS
OF 24S-T3 AND 75S-T6 ALUMINUM ALLOYS AND OF
SAE 4130 STEEL WITH STRESS-CONCENTRATION
FACTORS OF 2.0 AND 4.0

By H. J. Grover, S. M. Bishop, and L. R. Jackson

Battelle Memorial Institute



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SUMMARY

This report presents results of axial-load fatigue tests on notched specimens of three sheet materials: 24S-T3 and 75S-T6 aluminum alloys and normalized SAE 4130 steel. Notches included:

- (1) Stress-concentration factor 2.0: Central circular hole, symmetrical edge notches, and fillets
- (2) Stress-concentration factor 4.0: Symmetrical edge notches and fillets

For each type of specimen, fatigue tests were run at several levels of nominal mean stress, including a zero nominal mean stress.

Fatigue strengths for these notched specimens are compared with values previously reported for unnotched specimens of the same sheet materials.

INTRODUCTION

This is the second of a series of reports summarizing work on an investigation of the fatigue strengths of metals used in aircraft construction. The investigation, conducted at Battelle Memorial Institute, under the sponsorship and with the financial assistance of the National Advisory Committee for Aeronautics, has the objective of obtaining extensive basic data on the fatigue properties of three widely used sheet materials: 24S-T3 and 75S-T6 aluminum alloys and SAE 4130 steel.

The previous report (reference 1) presented data on unnotched specimens. Such data are of interest in regard to basic properties of the materials, but are not adequate for design of structural parts. Structures nearly always have necessary stress-raisers, such as holes, cutouts, or other sharp changes in section that are critical in fatigue. Accordingly, the investigation was extended to study the behavior of notched specimens of the three sheet materials. The present report presents the results of fatigue tests on sheet specimens with several types of notches.

The notch forms used were chosen in an effort to obtain systematic information covering several variables which might influence notch fatigue strength. Two notch severities, judged by the theoretical stress-concentration factor K_t , were used: $K_t=2.0$ and $K_t=4.0$. For the lower severity, notches of three different shapes (fillets, edge notches, and a circular hole) were selected; for the higher severity, two shapes (fillets and edge notches) were used. These different notch forms afford variation not only in stress-concentration factor but also in stress gradient and in volume of highly stressed material near the notch.

Some of the results presented in the first report are recapitulated in the present report to allow discussion of fatigue-strength reduction caused by the various notch forms.

The authors wish to thank Mr. Paul Kuhn, of the Structures Research Division of the Langley Aeronautical Laboratory of the NACA at Langley Field, Virginia, for his help and guidance during this investigation, and Mr. David O. Leeser, formerly on the staff of Battelle Memorial Institute, who did most of the experimental work described in this report.

EXPERIMENTAL PROCEDURES

Material

The materials used in this investigation were supplied from selected stock retained for this purpose at the Langley Aeronautical Laboratory of the NACA. Coupons were cut from 0.090-inch-thick commercial sheets of 24S-T3 and of 75S-T6 aluminum alloys and from 0.075-inch-thick commercial sheets of normalized SAE 4130 steel. Details of sheet layout are to be found in reference 1.

Static-strength properties, some of which are repeated from the previous report, are given in table 1.

Notched Specimens

Figures 1 and 2 show dimensional drawings of the notched specimens. Notch dimensions were chosen, on the basis of information available in the literature and unpublished information from the Langley Aeronautical Laboratory, to produce theoretical stress-concentration factors of 2.0 and 4.0. Notches were cut with tools specially machined to produce the contour desired in each case. Machining cuts were successively lighter, so that the depth of each of the last two cuts was about 0.0005 inch. Following machining, specimens were finished by electropolishing, which removed an average of about 0.0003 inch from the surface in the region of the notch and left a surface estimated to have about an 8-microinch profilometer value. Specimens were shadowgraphed after electropolishing: The dimensions and tolerances in figures 1 and 2 are those actually measured. It was estimated that errors in $\ensuremath{K_{t}}$ due to variation in notch dimensions were not greater than those due to uncertainties in theoretical and photoelastic information on which notch design was based.

Figures 3 and 4 show stress-coat patterns obtained on some of the notched specimens under tensile loading. These patterns indicate stress distributions such as would be expected from the theory of elasticity.

Table 1 contains static-failure strengths for the notched specimens. Nominal tensile stresses at failure were not greatly different from values of ultimate tensile strength determined by tests on standard tensile specimens. Thus, stress-concentration effects of the notches were apparently alleviated by plastic deformation before static failure occurred.

Fatigue Test Procedures

Fatigue tests were run on Krouse direct repeated-stress testing machines at speeds in the range 1100 to 1500 cycles per minute. A description of the machines is given in reference 1. It is estimated that precision of load measurement and maintenance was about ±3 percent in tension-tension tests. In tests involving reversal of load, sheet specimens were restrained from buckling by the use of guide plates. Estimation of precision of loading in such cases was indirect; it is believed that error in load value, in reversed-load testing, did not usually exceed ±5 percent.

FATIGUE TEST RESULTS

Results of axial-load fatigue tests are given in tables 2 to 7. These results are plotted in the form of S-N diagrams in figures 5 to 10. All stress values indicated on these diagrams are nominal net-area stresses. While the data are insufficient to afford a statistical evaluation of scatter, it may be noted that observed points fall closely on the S-N curves drawn.

Figures 11 to 25 show the same results plotted in another manner: As constant-lifetime diagrams of nominal stress amplitude plotted against nominal mean stress. In these diagrams, however, "points" are not directly observed values but are values read from the faired S-N curves in figures 5 to 10.

DISCUSSION OF RESULTS

Diagrams such as those shown in figures 11 to 25 have been suggested for use in design. However, the designer encounters a great variety of notch forms and will seldom be concerned with one exactly like any used in this or any other laboratory fatigue investigation. Consequently, there is considerable interest in attempting to understand the notch fatigue behavior of a material sufficiently that results of a limited number of tests may be generalized to apply to the great variety of notch forms and loading conditions encountered in aircraft service.

Tables 8 to 10 summarize fatigue-strength values for notched specimens, and include corresponding values previously reported for unnotched specimens of the same sheet materials. The following discussion is based on values listed in these summary tables.

In view of the local nature of fatigue failure, it is expected that the fatigue strength of a notched specimen will be strongly influenced by stress concentration at the root of the notch. Thus, it seems desirable to examine the results of notch fatigue tests in terms of estimated peak stress at the notch root. Conventionally, such examination is usually made in terms of a "fatigue-strength reduction factor," denoted by K_f , and defined as the ratio of unnotched fatigue strength to notched fatigue strength. While such a definition is unambiguous for test runs under fully reversed load, further specification is needed when the mean load differs from zero. At least three definitions of K_f have been advocated for such cases:

(1) Load-ratio definition,

 $K_{1} \equiv \frac{Maximum \ stress \ for \ unnotched \ specimen}{Nominal \ maximum \ stress \ for \ notched \ specimen \ at \ same \ load \ ratio \ and \ lifetime$

- (2) Load-amplitude definition,
- $K_f^{i} \equiv \frac{Stress \ amplitude \ for \ unnot ched \ specimen}{Nominal \ stress \ amplitude \ for \ not ched \ specimen \ at \ same \ nominal \ mean \ stress \ and \ lifetime$
 - (3) Maximum load at fixed mean load definition,
- Kf" = Maximum stress for unnotched specimen

 Nominal maximum stress for notched specimen at same nominal

 mean stress and lifetime

Calculations of K_f , K_f^i , and $K_f^{"}$, for the values listed in tables 8, 9, and 10, show that none of these three fatigue-strength reduction factors are constant for the full range of notch forms and stress levels covered in this investigation. In general, K_f and $K_f^{"}$ are less than K_t , while K_f^i sometimes exceeds K_t . Design, based on the approximation that any one of these K_f^i s is predictable from the theoretical stress-concentration factor of the notch or even upon the assumtpion that a K_f (like one of the three defined here) is constant over a range of stress levels, might be seriously in error.

Consideration of the probable peak stress at the notch root, in relation to the stress levels at which fatigue failures occur, indicates one reason for this lack of simple correlation between $K_{\rm f}$ and $K_{\rm t}$. For a great deal of the region of stress, plastic flow undoubtedly occurred at the point of highest stress. It will be convenient, in further discussion, to treat, first, low stress levels for which such plastic flow may be negligible and, second, the remaining part of the stress field investigated.

Region of Low Maximum Stress

In the region of low stresses and little plastic flow, the peak stress at the root of a notch should be given by $K_{\rm t}$ times the nominal stress to which the notched specimen is loaded. Thus, at low stress levels, one might expect, insofar as fatigue failure depends on maximum stress, that a notched specimen under a cycle from a nominal minimum stress $m_{\rm n}$ to a nominal maximum stress $M_{\rm n}$ would fail in the same lifetime as an unnotched specimen under a cycle from minimum stress $K_{\rm t}m_{\rm n}$ to maximum stress $K_{\rm t}m_{\rm n}$. In this case, the "load-ratio fatigue-strength reduction factor" $K_{\rm f}$ should equal $K_{\rm t}$ through this low-stress region.

Examination of tables 11, 12, and 13 shows that $K_{\hat{I}}$ usually approaches $K_{\hat{t}}$ in regions of low maximum stress. However, even in such regions, $K_{\hat{I}}$ is often less than $K_{\hat{t}}$, particularly for the more severe notches ($K_{\hat{t}} = 4$). Neuber (reference 2) suggests that departures from elastic theory should be expected in notches so sharp that stress gradients are large over regions of local inhomogeneity and anisotropy of the material under investigation. He proposes using a "technical stress-concentration factor," defined by

$$K_{\overline{N}} \approx 1 + \frac{K_{t} - 1}{1 + \sqrt{\frac{\rho^{i}}{\rho}}}$$
 (1)

In this defining equation, ρ is the radius of the notch, and $\rho^{\text{!}}$ is a constant (with the dimension of length) of the material. Neuber suggests that the value of $\rho^{\text{!}}$ may be about 0.02 inch for many materials. Table 14 shows values of K_N for the notches used in this investigation for several values of $\rho^{\text{!}}$. Comparing these values with values of K_f in tables 11, 12, and 13, it appears that K_f is often nearer to K_N (for $\rho^{\text{!}}$ = 0.02 in.) than to K_t . No other value of $\rho^{\text{!}}$ affords much better agreement for all values of K_f .

Thus, results obtained in low-stress-level fatigue tests on the notched sheet specimens are approximately predictable by assuming $K_{\mathrm{T}} = K_{\mathrm{N}}$. However, until the limitations of this assumption are more completely established, such predictions should be used with caution in design of notched parts.

Region of High Maximum Stress

Figure 26 is a schematic illustration of effects likely to occur in tension-tension tests in which the maximum stress is high enough to cause local yielding at the notch root. Two effects are to be noted:

- (1) At the top of the load cycle, the maximum local stress is less than $K_t M_n$; that is, local deformation alleviates the stress concentration predicted for ideally elastic material
- (2) Upon unloading to minimum load, residual stress at the notch root decreases the minimum local stress below that expected for ideally elastic behavior

Both of these effects have been observed experimentally (references 3 to 7). Some time ago, Hartmann (reference 8) suggested an approximate method of estimating the alleviation of the stress-concentration factor at maximum load; Stowell (reference 4) has recently suggested another method of approximating the local maximum stress. No satisfactory method of theoretically estimating the residual stress and the resultant minimum local stress has been reported.

For fully reversed loading through large stress amplitudes, additional complications may occur. In this case, local yielding may take place both during tensile loading and during compressive loading. Cumulative strain hardening may alter the behavior during successive load cycles. Behavior in this region is too complex for prediction on the basis of currently available information.

If this picture of fatigue-strength reduction at high stress levels is correct, it would be expected that the stress ratio would differ from the load ratio, and that $K_{\hat{I}}$ would differ from $K_{\hat{I}}$ (or from $K_{\hat{N}}$). In fact, it appears doubtful that any simply defined fatigue-strength reduction ratio could be expected to remain constant over the full range of stress levels. It is possible that detailed consideration of effects of plastic deformation and residual stresses will afford approximate rules useful in design.

CONCLUSIONS

Axial-load fatigue test results have been obtained on notched sheet specimens of 24S-T3 and 75S-T6 aluminum alloys and of SAE 4130 steel. Several notch forms were used and tests were run at several levels of mean stress. The results show that:

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- 1. Reduction in fatigue strength (in terms of nominal stresses) varies with:
 - (a) Notch severity (theoretical stress-concentration factor)
 - (b) Notch form, especially for severe notches
 - (c) Material
 - (d) Stress level both nominal mean stress and nominal stress amplitude
- 2. Simply defined fatigue-strength reduction factors do not appear to have useful correlation with the theoretical stress-concentration factor.

Battelle Memorial Institute Columbus, Ohio, August 15, 1950

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TABLE 1.- STATIC TENSILE AND COMPRESSIVE STRENGTHS OF SOME ALUMINUM

AND STEEL SHEET SPECIMENS USED IN FATIGUE TESTS

			Average tensile properties			Average compressive properties (a)	
Material	Grain direction	Type specimen	Elongation (percent)	Yield strength (psi)	Ultimate strength (psi)	Yield strength (psi)	Modulus of elasticity (psi)
248-T3 248-T3	With Cross	Unnotched Unnotched	18.2 18.3	54,000 50,000	73,000 71,000	44,500 50,000	10.65 × 10 ⁶ 10.45
248-T3 248-T3 248-T3	With With With	Hole-type notch (K _t = 2.0) Fillet-type notch (K _t = 2.0) Edge-cut notch (K _t = 2.0)			71,500 72,000 74,500		
248-T3 248-T3	With With	Fillet-type notch (Kt = 4.0) Edge-cut notch (Kt = 4.0)			65,900 65,400		
758-T6 758-T6	With Cross	Unnotched Unnotched	11.4 11.0	76,000 75,000	82,500 82,500	74,000 78,500	10.45 10.55
758-T6 758-T6 758-T6	With With With	Hole-type notch $(K_t = 2.0)$ Fillet-type notch $(K_t = 2.0)$ Edge-cut notch $(K_t = 2.0)$			80,500 82,500 87,500		
758-16 758-16	With With	Fillet-type notch (Kt = 4.0) Edge-cut notch (Kt = 4.0)			80,000 82,500		
4130 4130 4130	With Cross With	Unnotched Unnotched Hole-type notch (Kt = 2.0)	14.25 12.5	98,500 101,000	117,000 120,000 120,500	86,000 97,000	30.4 31.3
4130 4130	With With With	Fillet-type notch $(K_t = 2.0)$ Edge-cut notch $(K_t = 2.0)$ Fillet-type notch $(K_t = 4.0)$			119,000 117,000 119,000		
4130 4130	With	Edge-cut notch (Kt = 4.0)			129,000		

⁸Values for unnotched specimens taken from reference 1. All stress values nominal, based on original net section.

bSpecimen failed in grip section at load noted.

TABLE 2.- AXIAL-LOAD FATIGUE TEST RESULTS' FOR 24s-T3 ALUMINUM SHEET SPECIMENS; NOTCHED, $K_{\mbox{\scriptsize t}}=2.0$

Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks			
	(a) Nomi	nal mean stress, () psi			
	Но	le-type notch				
A2752C A6552C A1052C A952C A1552C A2052C A1852C	34,000 28,000 24,000 20,000 15,000 12,000 10,000	2,500 17,500 28,800 70,000 405,000 >10,907,000 >10,994,800	Did not fail Do.			
	. Ed	ge-cut notch				
A79S2B A84S2B A73S3B A80S3B A30S2B A88S2B A29S2B	35,000 35,000 30,000 30,000 28,000 25,000 20,000	3,400 3,500 6,500 7,700 	Buckled			
A73S2B A35S2B A40S2B A1S2B A74S3B	15,000 15,000 15,000 13,500 11,000	754,000 160,000 210,000 287,000 >10,586,000	Failed in grip Failed in flaw Did not fail			
	Fillet-type notch					
A4052A A3952A A7352A A3452A A7852A	30,000 22,500 18,000 15,000 12,000	10,000 64,000 233,000 500,000 5,251,800	·			

TABLE 2.- AXIAL-LOAD FATIGUE TEST RESULTS FOR 24S-T3 ALUMINUM SHEET SPECIMENS; NOTCHED, $K_{\rm t}$ = 2.0 - Continued

Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks (1)
	(b) Nomina	l mean stress, 10,	,000 psi
]	Hole-type notch	
A2882C A1282C A5382C A5182C A6382C A6282C A5482C A5582C A5982C A5782C A1482C A2482C	46,000 43,000 40,000 35,000 35,000 30,000 25,000 22,000 20,000 19,000 18,500	2,700 5,500 6,100 16,500 17,100 30,100 36,200 118,000 283,700 542,700 181,400 5,564,300 6,568,100	Failed in flaw
A2282C	18,000	>13,013,200	Did not fail
		Edge-cut notch	
A7583B A8783B A4782B A4782B A4682B A4582B A3282B A7482B A7982B A3882B A8382B A8382B	44,000 44,000 40,000 35,000 35,000 30,000 25,000 22,000 21,000 20,000 19,000	2,900 3,000 6,500 14,900 15,500 35,000 43,400 124,200 168,700 507,400 7,687,400 >15,018,800	Did not fail
	F	illet-type notch	
A8752A A7152A A4452A A8052A A7752A A3352A A8852A A8152A A8552A A8252A	43,000 40,000 35,000 35,000 30,000 30,000 25,000 22,500 22,000	4,000 6,500 27,100 30,000 • 73,200 75,100 129,200 288,100 283,200 >10,486,700	Did not fail

Unless otherwise noted, specimens failed at notch root in region of critical stress concentration.

TABLE 2.- AXIAL-LOAD FATIGUE TEST RESULTS FOR 24S-T3 ALUMINUM SHEET SPECIMENS; NOTCHED, $K_{\mbox{\scriptsize t}}=2.0$ - Continued

Specimen	Nominal maximum stress (ps1)	Life (cycles)	Remarks			
		l mean stress, 20,000) psi			
		Hole-type notch	····			
A2652C A1952C A1652C A6152C A9652C A5252C A10052C A9152C A4952C A5652C A9552C	52,500 52,500 52,500 49,000 45,000 40,000 35,000 31,000 29,500 29,500 27,500	2,300 4,000 3,000 7,100 15,600 35,600 75,400 213,200 2,319,700 9,536,000 >10,936,000	Did not fail			
		Edge-cut notch				
A84s2b A85s3b A70s2b A42s2b A91s2b A78s2b A80s2b A93s2b A90s2b A82s2b A85s2b A85s2b A85s2b A85s2b A85s2b A85s2b	52,500 49,000 49,000 45,000 45,000 35,000 35,000 31,500 31,000 29,500 27,500	3,100 9,300 6,000 21,800 25,300 48,300 66,500 82,200 28,200 128,500 218,700 >13,114,700 >15,671,300	Failed in flaw Failed lower grip Did not fail			
	Fillet-type notch					
A79S2A A83S2A A89S2A A40S2A A92S2A A96S2A A69B2A A70S2A A94S2A	52,500 49,000 45,000 40,000 35,000 31,500 29,500 29,500 27,500	4,500 8,300 19,800 30,300 67,000 595,400 1,042,400 >10,305,000 >12,693,800	Did not fail Do.			

TABLE 2.- AXIAL-LOAD FATIGUE TEST RESULTS FOR 24S-T3 ALUMINUM SHEET SPECIMENS; NOTCHED, $K_{\rm t}$ = 2.0 - Concluded

Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks (1)		
	(d) Nominal ma	ean stress, 30,0	000 psi		
	Hole	e-type notch			
A1782C A2382C A5082C A6782C A2182C A6682C A6482C A6882C	60,000 54,000 50,000 45,000 42,500 40,000 39,000 38,000	2,000 14,000 21,800 55,700 68,500 243,800 395,700 >12,000,600	Did not fail		
	Edge	e-cut notch			
A7053B A8153B A8352B A4452B A7153B A4852B A3452B A3652B A8752B A3752B	60,000 60,000 54,000 50,000 45,000 42,500 40,000 40,000 38,500	4,300 4,500 9,600 25,700 25,700 63,500 152,900 259,200 315,500 >10,537,100	Did not fail		
Fillet-type notch					
A2952A A3252A A4852A A4352A A3852A A4752A A3652A A4552A	54,000 50,000 45,000 45,000 42,500 40,000 39,000 38,000	17,600 29,200 47,000 95,300 114,600 173,200 >10,608,000 >11,541,600	Did not fail Do.		

Unless otherwise noted, specimens failed at notch root in region of critical stress concentration.

TABLE 3.- AXIAL-LOAD FATIGUE TEST RESULTS FOR 24S-T3 ALUMINUM SHEET SPECIMENS; NOTCHED, Kt = 4.0

Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks (1)		
<u></u>	(a) Nomina	l mean stress, O psi			
	Edge	-cut notch			
A1083B A4783B A983B A583B A3483B A4483B A4383B A5083B	22,500 17,500 12,500 10,000 7,500 7,000 8,000 5,000	3,200 10,000 53,400 121,500 1,256,700 6,309,100 944,400 >11,169,000	Did not fail		
	Fille	t-type notch			
Alisa Alésa Agosa Al4sa Alésa A50sa Abosa	25,000 20,000 15,500 12,500 9,500 9,500 7,500	4,400 15,000 38,500 140,100 1,066,000 548,700 >10,969,000	Did not fail		
·	(b) Nominal me	an stress, 10,000 psi			
	Edge	-cut notch			
A2683B A3883B A3183B A4583B A4083B A3383B A2483B A2183B A1583B A1583B A383B A383B A2883B A1183B	30,000 30,000 27,500 27,500 25,000 22,500 20,000 20,000 17,500 16,500 15,000	2,000 4,000 3,000 5,700 12,000 26,000 52,000 62,500 71,000 61,500 112,000 >10,533,800 >10,408,300	Did not fail Do.		
Fillet-type notch					
A4183A A1983A A3983A A983A A283A A383A A4483A A4283A A683A	35,000 32,500 32,500 30,000 27,500 25,000 22,500 20,000 17,500	2,500 3,100 2,800 6,500 12,000 25,500 49,800 87,000 653,700 >10,733,000	Did not fail		

Table 3.- Axial-load fatigue test results for 24s-t3 aluminum sheet specimens; notched, $K_{\rm t}$ = 4.0 - Concluded

Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks			
	(c) Nominal me	ean stress, 20,000 psi				
	Edge	-cut notch				
A1283B A2983B A4983B A1683B A3783B A1383B	35,000 32,500 30,000 27,500 25,000 22,500	3,700 9,000 26,600 39,400 1,343,000 >10,321,500	Did not fail			
	Fille	et-type notch				
A1753A A4853A A3553A A3853A A2553A A453A A2253A A553A A5953A A1953A A1053A A3353A	40,000 37,500 35,000 32,500 30,000 27,500 27,500 27,500 25,000 25,000 22,500	4,000 6,000 10,200 15,500 21,000 44,500 69,800 80,000 161,500 300,000 5,797,000 >10,213,000	Did not fail			
	(d) Nominal me	an stress, 30,000 psi				
	Edge	-cut notch				
A4683B A4283B A1783B A3283B A2083B A3983B A3083B	47,500 45,000 42,500 40,000 37,500 35,000 32,500	2,200 4,000 7,000 14,000 24,500 124,500 >10,450,000	Did not fail			
	Fillet-type notch					
A2183A A853A A3183A A2983A A2463A A4763A A1583A	47,500 45,000 42,500 40,000 37,500 35,000 32,500	3,300 7,500 11,500 26,700 61,000 413,700 >10,703,000	Did not fail			

TABLE 4.- AXIAL-LOAD FATIGUE TEST RESULTS FOR 75S-T6 ALUMINUM SHEET SPECIMENS; NOTCHED, $\kappa_{\rm t}$ = 2.0

Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks
	(a) Nomin	nal mean stress, O ps	L
	Но	le-type notch	
B7982C B10082C B9982C B9482C B9582C B8682C B9182C	36,000 34,000 28,000 24,000 21,000 18,000 16,000	3,400 3,200 14,000 42,000 86,000 412,400 1,028,000	
	Edg	ge-cut notch	
B100S3B B95S3B B50S2B B93S3B B92S3B B92S3B B47S2B B44S2B B45S3B B26S2B B6S2B B17S2B B28S2B B17S2B B28S2B B17S2B B10S2B B43S2B	34,000 34,000 34,000 30,000 30,000 28,000 24,000 21,000 18,000 15,000 15,000 12,500	5,500 5,400 4,000 12,000 11,400 19,000 23,700 89,000 213,000 347,500 579,000 1,564,300 >10,853,500	Buckled Failed in grip Did not fail
	F11.	let-type notch	
B45S2A B42S2A B35S2A B28S2A B30S2A B10S2A B17S2A B17S2A	34,000 34,000 34,000 31,000 31,000 28,000 28,000 24,000 21,000	10,000 11,500 	Buckled Buckled
B2382A B14S2A	18,000 15,000	115,000 4,541,800	

TABLE 4.- AXIAL-LOAD FATIGUE TEST RESULTS FOR 758-T6 ALUMINUM SHEET SPECIMENS; NOTCHED, $\rm\,K_{t}\,=\,2.0\,$ - Continued

Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks		
	(c) Nomin	al mean stress, 20	0,000 psi		
		Hole-type notch			
B7452C 56,000 B8552C 55,000 B4852C 50,000 B5852C 45,000 B5752C 40,000 B7252C 35,000 B6652C 32,000 B9752C 30,000 B9352C 29,000 B9652C 28,000		2,200 3,000 5,400 9,300 12,000 29,500 46,000 165,600 536,100 >11,250,000	Did not fail		
		Edge-cut notch			
B21S2B B97S3B B3S2B B14S2B B40S2B B11S2B B12S2B B27S2B B27S2B B23S2B B18S2B	56,000 54,000 50,000 45,000 40,000 35,000 32,500 30,000 29,000 28,000	2,100 3,200 5,000 11,500 13,400 28,000 76,800 621,900 >284,000 >10,781,700	Failed above lower grip Did not fail		
Fillet-type notch					
B1152A B652A B152A B252A B1952A B4352A B1352A B3452A	54,000 50,000 45,000 40,000 35,000 32,500 30,000 29,000	5,400 9,000 17,500 18,500 33,500 53,000 105,000 >10,249,900	Did not fail		

Unless otherwise noted, specimens failed in notch root in region of critical stress concentration.

TABLE 4.- AXIAL-LOAD FATIGUE TEST RESULTS FOR 75S-T6 ALUMINUM SHEET SPECIMENS; NOTCHED, $K_{\rm t}$ = 2.0 - Continued

Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks			
	· (b) Nomin	al mean stress, 10	,000 psi			
		Hole-type notch				
B73S2C B75S2C B71S2C B60S2C B65S2C B70S2C B92S2C B68S2C B59S2C B69S2C	46,600 46,500 45,000 40,000 35,000 30,000 25,000 22,000 20,500 20,000	2,600 2,700 3,100 6,800 13,000 22,500 60,700 227,700 >12,710,400 >10,547,800	Did not fail Do.			
		Edge-cut notch				
B4S2B B25S2B B19S2B B7S2B B29S2B B12S2B B33S2B B34S2B B5S2B	45,000 40,000 35,000 30,000 25,000 23,500 22,500 22,500 20,500	3,000 7,000 18,500 46,200 242,000 2,678,600 627,500 >10,581,900 >12,653,200	Failed in upper grip Did not fail Do.			
	Fillet-type notch					
B12S2A B31S2A B44S2A B41S2A B7S2A B16S2A B26S2A B27S2A	45,750 40,000 35,000 30,000 25,000 22,500 22,500 21,000	5,800 13,500 20,500 59,900 189,600 718,500 2,998,000 >10,336,900	Failed in upper grip Did not fail			

TABLE 4.- AXIAL-LOAD FATIGUE TEST RESULTS FOR 758-T6 ALUMINUM SHEET SPECIMENS; NOTCHED, $K_{\rm t}$ = 2.0 - Concluded

Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks				
	(d) Nominal mean stress, 30,000 psi.						
		Hole-type notch					
B6382C B7682C B6482C B8382C B9782C B8782C B8882C B9882C B6282C B6182C B5682C	68,000 66,100 65,000 60,000 55,000 45,000 42,500 39,000 38,000 37,000	1,800 2,400 2,200 5,200 7,500 12,000 24,800 42,800 198,200 527,300 >10,112,300	Did not fail				
		Edge-cut notch					
B39S2B B53S2B B22S2B B45S2B B31S2B B30S2B B36S2B B35S2B B15S2B B16S2B	66,500 63,000 60,000 55,000 50,000 45,000 42,500 39,000 38,000 37,000	2,800 2,300 4,100 8,300 12,500 24,000 35,000 81,000 >10,062,700 >10,363,600	Did not fail				
Fillet-type notch							
B3S2A B15S2A B4S2A B2OS2A B36S2A B38S2A B39S2A B5S2A	65,000 60,000 55,000 50,000 45,000 42,500 40,000 38,000	4,800 8,000 8,700 11,500 27,000 36,000 89,000 >9,978,500	Failed in upper grip				

Unless otherwise noted, specimens failed in notch root in region of critical stress concentration.

			
	Nominal	710.	
Specimen	maximum stress	Life (cycles)	Remarks
	(psi)	(Cycles)	(1)
<u> </u>	(PBI)	<u> </u>	(2)
	(a) Nomina	al mean stress, O psi	
	Edge	e-cut notch	
B4583B	20,000	5,300	
Blos3B	16,250	17,800	
В3583В	12,500	70,000	
B36S3B	9,250	339,200	1
B1983B B2883B	8,500 7,500	969,200	1
B20S3B	7,500	1,652,300 4,722,000	
B31S3B	5,500	>12,405,300	Did not fail
B2983B	4,000	>10,247,800	Do.
	Fille	et-type notch	
B3S3A	22,500	8,200	
B4853A	20,000	17,000	
B2683A	16,250	63,500	
B3983A	12,500	182,000	
B3183A	10,000	4,400,000	
B4383A	9,000	3,097,100	Failed in flaw
B4583A	7,500	>10,244,500	Did not fail
	(b) Nominal m	ean stress, 10,000 psi	
	Edge	e-cut notch	
в1683в	30,000	2,000	1
в2633в ,	25,000	8,000	1
B38S3B	22,500	13,000	1
B1783B	20,000	41,000	1
в2383в	20,000	39,000	
B183B	20,000	32,000	,
B483B	17,500	48,500	Budled to Place
B4183B	15,000 15,000	89,000 9,610,300	Failed in flaw
B4383B	12,500	>12,281,609	Did not fail
		et-type notch	· L
ni Eggs			
B1583A	30,000	4,000	1
B3883A B583A	27,500 25,000	10,000 14,500	\
B453A	22,500	45,800	1
B2183A	22,500	39,500	}
B1683A	20,000	39,000	Buckled
B4983A	20,000	43,000	Do.
B3383A	20,000	140,000]
B3783A	20,000	82,500	1
B13S3A	17,500	1,676,000	
B2783A	15,000	>10,000,000	Did not fail

TABLE 5.- AXIAL-LOAD FATIGUE TEST RESULTS FOR 758-T6 ALUMINUM

SHEET SPECIMENS; NOTCHED, $K_t = 4.0 - Concluded$

Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks								
	(c) Nominal mean stress, 20,000 psi										
	Edge-cut notch										
B2183B B2583B B1183B B983B B3783B B4883B B683B B4083B	35,000 32,500 30,000 30,000 27,500 25,000 22,500	2,500 5,500 10,500 10,700 16,800 46,500 566,500 >10,457,000	Did not fail								
	F111c	et-type notch									
B2953A B4753A B2553A B4053A B3453A B2453A	35,000 32,500 30,000 27,500 25,000 22,500	4,000 9,800 18,700 31,000 467,000 >9,475,000 ean stress, 30,000 psi	Did not fail								
		e-cut notch									
B783B B2583B B1383B B1483B B383B B4783B	42,500 40,000 40,000 37,500 35,000 32,500	4,000 10,000 7,800 15,000 32,700 >10,744,000	Did not fail								
	Fille	et-type notch									
B1783A B3253A B1453A B953A B2083A B1153A	45,000 42,500 40,000 37,500 35,000 32,500	3,500 6,300 12,300 22,000 119,000 >10,000,000	Did not fail								

Unless otherwise noted, specimens failed at notch root in region of critical stress concentration.

TABLE 6.- AXIAL-LOAD FATIGUE TEST RESULTS FOR SAE 4130 STEEL SHEET SPECIMENS; NOTCHED, $K_{\rm t}$ = 2.0

Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks (1)							
	(a) Nominal mean stress, O psi									
	Но	le-type notch								
C6252C C6152C C6052C C9952C C5652C C5952C	45,000 38,000 32,000 28,500 25,000 25,000	40,700 86,000 291,000 1,083,600 >11,429,000 >12,347,800	.Did not fail Do.							
	Ed	ge-cut notch								
C51S2B C197S2B C82B C15S2B C9S2B C13S2B C32S2B C14S2B C45S2B C33S2B C47S2B C47S2B C50S2B C49S2B	50,000 50,000 45,000 45,000 45,000 38,000 32,000 28,500 25,000 27,000 50,000 50,000	27,000 35,000 43,000 	Failed in grip Specimen buckled Did not fail Do. Failed in grip Do.							
	Fill	et-type notch								
C40S2A C29S2A C42S2A C25S2A C26S2A C13S2A	45,000 38,000 32,000 28,000 25,000 50,000	53,000 147,600 628,500 1,616,000 >10,468,400 29,000	Did not fail							

TABLE 6.- AXIAL-LOAD FATIGUE TEST RESULTS FOR SAE 4130 STEEL SHEET SPECIMENS; NOTCHED, $K_{
m t}=2.0$ - Continued

Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks						
	(b) Nominal mean stress, 10,000 psi								
	H	ole-type notch							
C86S2C C100S2C C92S2C C84S2C C63S2C C94S2C C83S2C	57,500 55,000 50,000 45,000 40,000 37,500	30,700 34,000 98,000 222,000 822,000 1,452,700 >10,043,000	Did not fail						
	E	lge-cut notch							
C46s2B C48s2B CXXXXS2B C43s2B C31s2B C30s2B CXS2B CXS2B C4s2B C39s2B C38s2B C44s2B C35s2B C31s2B C2s2B	60,000 60,000 57,500 57,500 55,000 50,000 45,000 40,000 40,000 37,500 37,500 35,000	28,500 31,300 31,500 31,700 55,900 93,000 151,000 255,000 290,000 421,000 900,000 1,101,600 540,000 >10,608,600	Failed in flaw Did not fail						
	Fi	llet-type notch							
C3452A C1152A C1652A C3252A C2852A C1052A C2752A C252A	60,000 57,500 55,000 50,000 45,000 40,000 37,500 35,000	36,000 45,800 103,000 235,000 545,000 1,157,000 >10,497,300	Specimen buckled Did not fail						

TABLE 6.- AXTAL-LOAD FATIGUE TEST RESULTS FOR SAE 4130 STEEL SHEET SPECIMENS; NOTCHED, $K_{\rm t}$ = 2.0 - Continued

Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks (1)						
	(c) Nominal mean stress, 20,000 psi								
	н(ole-type notch							
C6782C C7882C C6682C C6882C C8182C C7582C C7682C	65,000 60,000 55,000 50,000 47,500 45,000 42,500	31,000 62,000 175,000 517,000 857,600 1,184,700 >10,240,400	Did not fail						
	E	lge-cut notch							
C199S2B C189S2B C34S2B C22S2B C27S2B C20S2B C11S2B C36S2B C7S2B C12S2B C42S2B	72,500 70,000 70,000 65,000 60,000 55,000 50,000 47,500 45,000 42,500	18,000 24,500 28,000 39,700 70,900 227,000 535,900 1,002,000 1,557,700 >1,528,000 >10,480,300	Failed in upper grip Did not fail						
	F1.	llet-type notch							
C35S2A C24S2A C8S2A C36S2A C41S2A C15S2A C38S2A C17S2A C5S2A C22S2A	70,000 65,000 60,000 55,000 50,000 47,500 47,500 45,000 45,000	33,300 48,000 102,000 296,000 708,000 884,900 583,000 821,500 >10,583,700 >11,013,000	Failed in pit Do. Did not fail Do.						

Unless otherwise noted, specimens failed at notch root in region of critical stress concentration.

TABLE 6.- AXIAL-LOAD FATIGUE TEST RESULTS FOR SAE 4130 STEEL SHEET SPECIMENS; NOTCHED, $K_{\rm t}=2.0$ - Concluded

Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks (1)							
(d) Nominal mean stress, 30,000 psi										
	Hole-type notch									
079820 097820 077820 070820 074820 072820 080820 073820	75,000 75,000 70,000 65,000 60,000 60,000 55,000 52,500	36,000 38,500 60,000 128,000 287,600 104,800 610,600 >10,824,600	1/2-in. crack in flaw Did not fail							
		Edge-cut notch								
C37S2B C188S2B C194S2B C29S2B C28S2B C18S2B C24S2B C33S2B C19S2B	80,000 80,000 80,000 75,000 70,000 65,000 60,000 57,500 55,000	26,000 27,800 28,600 38,000 58,000 151,600 402,400 >10,262,800 >10,218,900	Did not fail Did not fail							
		Fillet-type notch								
C4152A C1852A C3152A C3352A C3052A C952A C2352A C452A C3952A C2152A C4352A	75,000 75,000 75,000 70,000 65,000 60,000 57,500 57,500 57,500 57,500 57,500	31,000 21,000 46,500 80,000 138,000 357,000 179,100 252,400 333,700 289,700 >10,729,000	Failed in flaw Did not fail							

Unless otherwise noted, specimens failed at notch root in region of critical stress concentration.

Table 7.- Axial-load fatigue test results for sae 4130 steel sheet specimens; notched, ' $\kappa_{\rm t}$ = 4.0

Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks (1)							
	(a) Nominal mean stress, O psi									
	Edge-cut notch									
C14982B C10482B C11182B C14482B C13082B C12582B C3882B C11582B C11582B C12282B C14282B C14282B	42,500 42,500 37,500 37,000 32,500 27,500 27,500 27,500 17,500 15,000 12,500	5,400 14,800 19,700 19,000 30,500 107,000 94,300 269,000 537,900 1,719,000 >10,325,000	Specimen buckled Did not fail							
	Fil	let-type notch								
C19083Z C18983A C19483A C19183A C5482A C13382A C4882A C20082A C11182A C4982A	C19083Z 45,000 C18983A 45,000 C19483A 40,000 C19183A 40,000 C5482A 35,000 C13382A 32,500 C4882A 30,000 C20082A 25,000 C11182A 20,000 C4982A 17,500		Specimen buckled Specimen buckled Did not fail							
	(b) Nominal	mean stress, 10,000 psi								
	Eć	ige-cut notch								
C14382B C11182B C10882B C11382B C10782B C14582B C10182B C13882B C13482B C13482B	50,000 50,000 42,500 38,750 35,000 30,000 27,500 27,500 25,000 22,500	12,600 9,000 30,700 37,000 82,000 197,000 223,000 344,000 740,000 >10,037,000	Did not fail							
	Fi	Llet-type notch								
C10482A C14882A C14582A C18383A C13383A C14982A C13982A C10882A C10382A	50,000 42,500 38,750 37,500 35,000 32,500 30,000 27,500 25,000	19,000 52,000 84,500 92,500 158,000 334,000 570,000 1,425,000 >10,327,800	Did not fail							

 $^{\mbox{$\frac{1}{2}$}}\!\mbox{Unless otherwise noted, specimens failed at notch root in region of critical stress concentration.$

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Table 7.- Axial-load fatigue test results for sae 4130 steel Sheet specimens; notched, $K_{\rm t}$ = 4.0 - Concluded

										
Specimen	Nominal maximum stress (psi)	Life (cycles)	Remarks							
	(c) Nominal mean stress, 20,000 psi									
	Edge-cut notch									
C11252B C14752B C13752B C12952B C13752B C10652B C11852B C15052B C10552B C10352B C13282B	57,500 55,000 51,250 45,000 40,000 37,500 35,000 35,000 35,000 32,500 32,500	11,400 18,000 27,000 59,000 106,000 134,000 202,000 181,500 164,000 >10,287,000 >10,000,000	Did not fail Do.							
	Fill	et-type notch								
C114S2A C129S2A C109S2A C120S2A C196S2A C127S2A C131S2A	57,500 51,250 45,000 40,000 40,000 37,500 35,000	23,000 53,300 106,000 449,000 225,000 433,000 >10,041,000	Did not fail							
	(d) Nominal m	ean stress, 30,000 psi	_ 							
	Edg	e-cut notch								
C12182B C12682B C14182B C13982B C12482B C13182B C13682B C11482B	65,000 60,000 55,000 52,500 50,000 47,500 45,000 42,500	10,000 16,500 26,500 43,000 64,000 100,000 262,000 ≥10,035,500	Did not fail							
	F111	et-type notch								
C13082A C15082A C13582A C12482A C12282A C12182A C12382A	72,500 65,000 60,000 55,000 50,000 47,500 45,000	9,000 22,500 34,500 80,000 168,000 173,000 >12,105,000	Did not fail							

TABLE 8.- SUMMARY OF NOTCH FATIGUE TEST RESULTS FOR 24s-T3 ALUMINUM SHEET SPECIMENS

Nominal mean	Notch			Nominal maximum stress (psi) for lifetimes (cycles) of -						
stress (psi)	type	Kt	103	5 x.10 ³	10 ⁴	5 × 10 ⁴	105	5 x 10 ⁵	106	107
0 × 10 ³	None Hole Edge Fillet Edge Fillet	122244	(40) × 10 ³ (26) (30)	5 ¹ × 10 ³ 33 33 (35) 21 2 ⁴	50 × 10 ³ 29.5 29.5 32 18 22	42 × 10 ³ 21 21 24 12.5 13.5	34 × 10 ³ 16.5 16.5 19 10 12	28 × 10 ³ 15 15 15 16 17 18 10	24 × 10 ³ 14 14 14 7.5 9.5	22 × 10 ³ 12 12 12 12 7 9
10	None Hole Edge Fillet Edge Fillet	12224	(50) (50) (50) (32.5) (37)	42 42 43 28.5 32	60 37 38 39 25 28	47 28 29 32.5 20 22	41 25.5 25.5 28 16 20	32 21.5 22.5 15.5 17.5	30.5 20.5 21 22 15 16.5	29 20 21 21 15 16
20	None Hole Edge Fillet Edge Fillet	1 2 2 2 4 4	(55) (60) (38) (43)	52 52 53 35 38	65 47 48 48 32 35	53 37 38 38 27 29	46 34 34 36 25 27	39.5 30 30 31 25 25.5	39 29 30 30•5 24 25	38 28 30 30 24 25
30	None Hole Edge Fillet Edge Fillet	1 W W W 4 4	(60) (48) (50)	(58) 59 (60) 45 46.5	70 55 56 57 41 43	59 45 47 47 36 38	54 42 43 44 35 36	48 39 39·5 39·5 3 ⁴ 35	47 38 39 39 34 34•5	46 38 39 39 34 34

Parentheses indicate value obtained by extrapolation.



TABLE 9.- SUMMARY OF NOTCH FATIGUE TEST RESULTS FOR 758-T6 ALUMINUM SHEET SPECIMENS

Nominal mean	Notch	Кt		Nominal maximum stress (psi) for lifetimes (cycles) of -						
stress (ps1)	type	n-t	103	5 × 10 ³	10 ⁴	5 × 10 ¹ 4	10 ⁵	5 x 10 ⁵	106	107
0 × 10 ³	None Hole Edge Fillet Edge Fillet	1 2 2 2 4 4	(37) × 10 ³ (40) (23) (26)	33 × 10 ³ 35 (37) 20 23.5	53 × 10 ³ 30 31 34 17 22	41 × 10 ³ 24 24 24 24 13 16.5	35 × 10 ³ 20 20 20 20 11 14	32.5 × 10 ³ 17 17.5 17.5 8.5 11	32 × 10 ³ 16.5 16.5 16.5 8	30 × 10 ³ 15.5 15.5 15.5 7.5 9.5
10	None Hole Edge Fillet Edge Fillet	1 2 2 4 4	(50) (50) (52) (32) (34)	42 42 45 27 29	62 37 38 42 23 26	47 26 29.5 30 18 22	40 24 26.5 26.5 16 20	39 22 24.5 26.5 15 18	36 21 23.5 23.5 14 17	35 21 23 23 14 17
20	None Hole Edge Fillet Edge Fillet	1 2 2 2 4 4	(60) (60) (62) (37) (38)	50 50 55 33 34	70 45 46 49 31 32	52 32.5 33 33 25 27	45 31 32 32 24 26	43 29.5 30 30 23 24.5	42 29 29•5 30 23 24	41 29 29.5 30 23 24
30	None Hole Edge Fillet Edge Fillet	1 2 2 4 4	(70) (70) (70) (70) (47) (48)	59.5 59.5 65 42 43	75 52 53 54 39	58.5 42 42 42 34 36	54 39•5 39•5 40 34 35	50 37.5 38.5 38.5 33. 34	49 37 38.5 38.5 33 34	49 37 38.5 38.5 33 34

 $^{^{\}mathbf{l}}$ Parentheses indicate value obtained by extrapolation.

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TABLE 10.- SUMMARY OF NOTCH FATIGUE TEST RESULTS FOR SAE 4130 STEEL SHEET SPECIMENS

Nominal mean	Notch	κ _t	No	minal maximum	stress ^l (psi) :	for lifetimes	(cycles) of -	
stress (psi)	type	nt.	10 ¹ 4	5 × 10 ⁴	10 ⁵	5 × 10 ⁵	106	107
0 × 10 ³	None Hole Edge Fillet Edge Fillet	1 2 2 2 4 4	75 × 10 ³ (53) (55) (56) 45 ⁴⁷	65 × 10 ³ 42 44.5 45 32 35	63 × 10 ³ 38 40 40 27 31	55 × 10 ³ 31 33 33 19 23	52 x 10 ³ 28 30 30 16 20	47 × 10 ³ 25 27 27 27 14 17
10	None Hole Edge Fillet Edge Fillet	1 2 2 4 4	87 (63) (64) (66) 52 57	79 54 54 55 38 43	73 50 50 51 34 38	68 41.5 41.5 42 25 31	60 39 39 39.5 23 27.5	60 35 37 37 23 26
. 20	None Hole Edge Fillet Edge Fillet	1 2 2 4 4	95 (74) (76) (80) 58 63	87 65 65 66 45 51	· 81 59 60 61 41 46	75 50 50 51 34 37•5	68 46 47 47 34 36	68 44 45 47 33 36
30	None Hole Edge Fillet Edge Fillet	7 5 7 7	103 (84) (85) (87) 6 ⁴ 72	93 71 72 73 52 57	89 88 89 99 19 53	82 57 58 58 44 46	76 55 57 57 44 46	76 55 57 57 43 46

 $^{^{1}}$ Parentheses indicate value obtained by extrapolation.

TABLE 11.- FATIGUE-STRENGTH REDUCTION FACTORS

FOR 24S-T3 ALUMINUM SHEET SPECIMENS

AT LOW STRESS LEVELS

Notch type	Load ratio		Fatigue-strength reduction factor, K _f , at lifetimes (cycles) of -								
		5 × 10 ⁴	10 ⁵	5 × 10 ⁵	106	107					
	(a) For notches having K _t = 2.0 and for nominal maximum stress <27,000 psi										
Hole Edge Fillet Hole Edge Fillet	-1.00 -1.00 -1.00 50 50	2.0 2.0 1.8	2.1 2.1 1.8 1.9 1.9	1.9 1.9 1.8 1.8	1.8 1.8 1.8 1.8 1.8	1.8 1.8 1.8 1.8 1.8					
	(b) For notches having K _t = 4.0 and for nominal maximum stress <13,500 psi										
Edge Fillet Edge Fillet	-1.00 -1.00 50 50	3.4 ·3.1 	3.4 2.8 3.3	3.5 2.8 3.2 2.4	3.2 2.5 3.4 2.4	3.1 2.4 3.1 2.4					

TABLE 12.- FATIGUE-STRENGTH REDUCTION FACTORS

FOR 75S-T6 ALUMINUM SHEET SPECIMENS

AT LOW STRESS LEVELS

Notch type	Load ratio		Fatigue-strength reduction factor, K _f , at lifetimes (cycles) of -							
		104	5 × 10 ⁴	10 ⁵	5 × 10 ⁵	106	107			
			naving K mum stres	•	.0 and f ,000 psi	or				
Hole. Edge Fillet Hole Edge Fillet	-1.00 -1.00 -1.00 50 50	1.7 1.7 1.8 1.9 1.9 1 1.6 1.7 1.8 1.9 1.9 1 1.8 1.9 1.9 1 1 1.7 1.8 1.8 1 1					1.9 1.9 1.9 1.9 1.8			
(b)	(b) For notches having $K_t = 4.0$ and for nominal maximum stress < 19,000 psi									
Edge Fillet Edge Fillet	-1.00 -1.00 50 50	3.1	3.2 2.5 3.0 2.6	3.2 2.5 3.3 2.6	3.8 2.9 3.8 2.8	4.0 3.1 4.0 3.0	4.0 3.2 4.0 3.1			

TABLE 13.- FATIGUE-STRENGTH REDUCTION FACTORS

FOR SAE 4130 STEEL SHEET SPECIMENS

AT LOW STRESS LEVELS

Notch type	Load ratio	Fatigue-strength reduction factor, K _f , at lifetimes (cycles) of -								
		5 × 10 ⁴	105	5 × 10 ⁵	106	107				
(a) For notches having K _t = 2.0 and for nominal maximum stress <49,250 psi										
Hole Edge Fillet Hole Edge Fillet	-1.00 -1.00 -1.00 50 50	1.5 1.5 1.4 	1.7 1.6 1.6 1.6 1.6	1.8 1.7 1.7 1.8 1.7	1.9 1.7 1.7 1.9 1.9	1.9 1.7 1.7 1.8 1.8				
(b) For notches having K _t = 4.0 and for nominal maximum stress < 24,625 psi										
Edge Fillet Edge Fillet	-1.00 -1.00 50 50			2.9 2.4 3.0	3.3 2.6 3.2 2.6	3.4 2.8 3.5 2.8				



Table 14.- values of $\ensuremath{\,\kappa_{\!N}}$ for several assumed values of $\ensuremath{\,\rho^{\,\mathfrak{k}}}$

Type of Notch	Kt	Radius, p (in.)	Value 1 of Neuber's "technical stress-concentration factor," $K_{ m N}$, for -				
			ρ' = 0.02 in.	$\rho^{\dagger} = 0.01 \text{ in.}$	$\rho^{i} = 0.005 \text{ in.}$	$\rho^t = 0.001$ in.	
Hole	2 2 2	1.5000	1.90	1.95	1.95	1.95	
Edge		.3175	1.80	1.85	1.90	1.95	
Fillet		.1736	1.75	1.80	1.85	1.95	
Edge	4	.0570	2.90	3.10	3.30	3.65	
Fillet	4	.0195	2.50	2.75	3.00	3.45	

 1 Computed, to the nearest 0.05, from the relation:



$$K_{N} = 1 + \frac{K_{t} - 1}{1 + \sqrt{\rho^{t}/\rho}}$$

Neuber suggests $\rho^{\dagger} = 0.0189$ in.

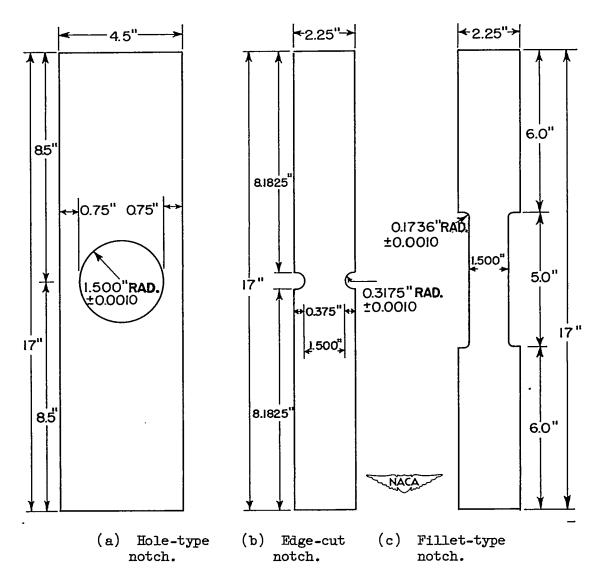


Figure 1.- Notched fatigue test specimens with K_t = 2.0.

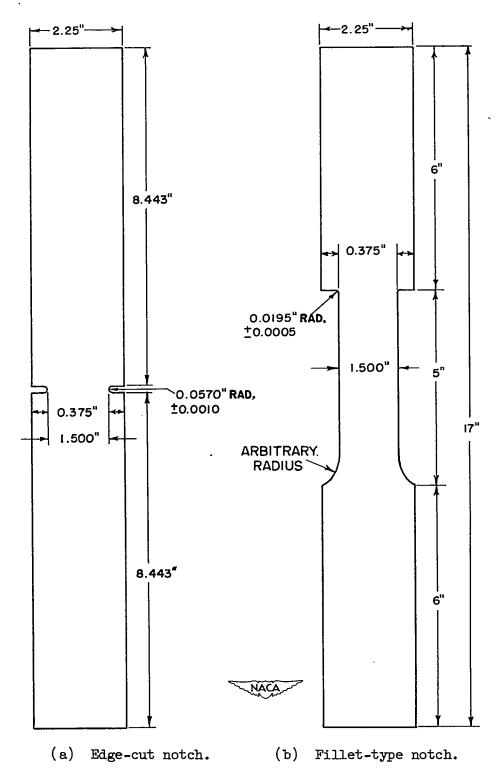


Figure 2.- Notched fatigue test specimens with $K_{t} = 4.0$.

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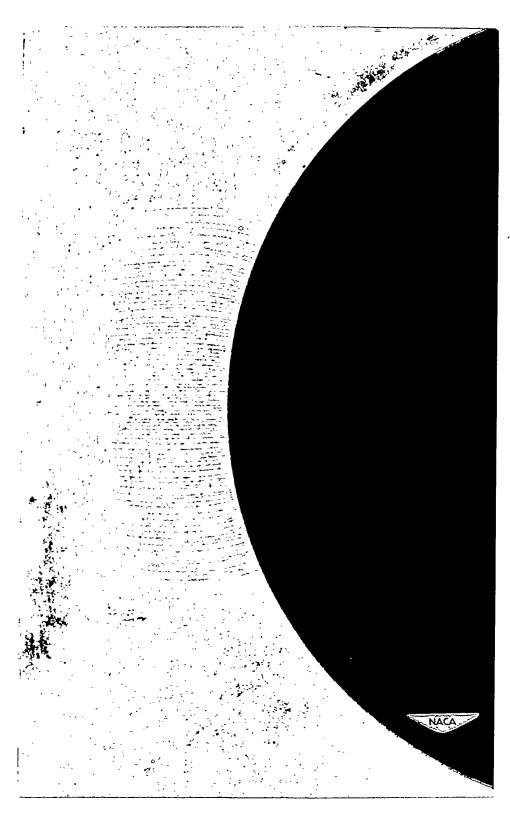


Figure 3.- Stress-coat pattern obtained at approximately 15,000 psi. Nominal loading of hole-type notch. $K_{\rm t}$ = 2.0. Approximately 3X.

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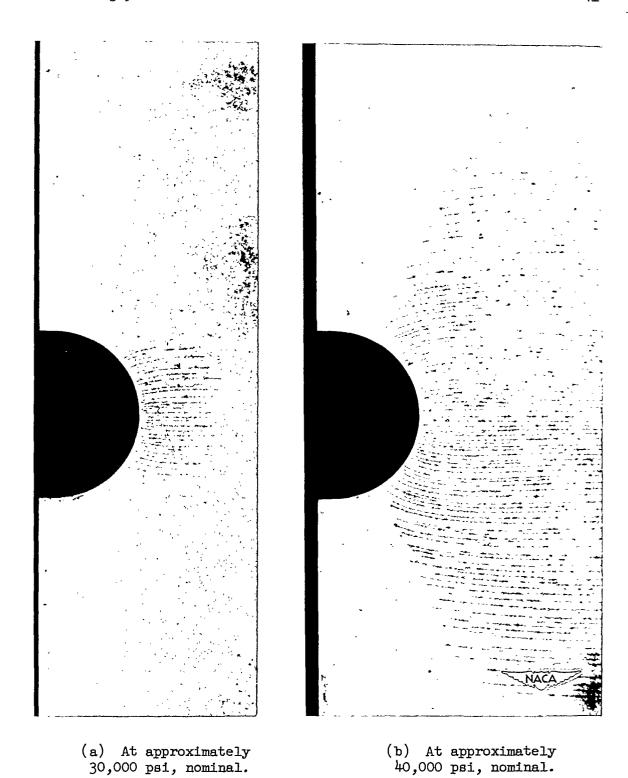


Figure 4.- Stress-coat patterns obtained on edge-cut notch. $K_t = 2.0$. Approximately 3X.

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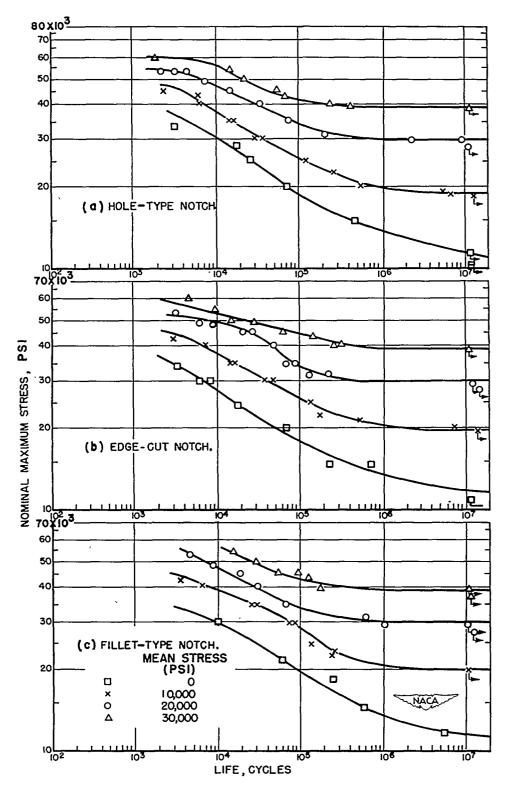


Figure 5.- Results of axial-load fatigue tests on notched 24S-T3 aluminum sheet specimens. $K_{\mbox{\scriptsize t}}$ = 2.0.

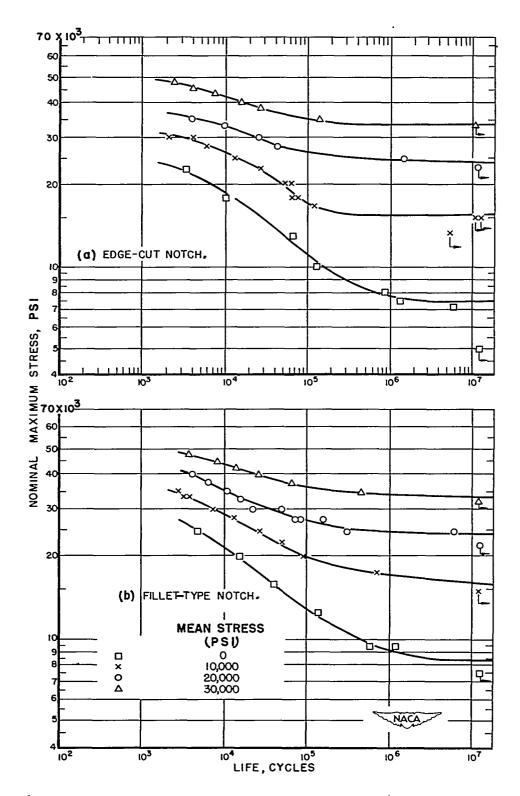


Figure 6.- Results of axial-load fatigue tests on notched 24S-T3 aluminum sheet specimens. $K_{\mbox{\scriptsize t}}$ = 4.0.

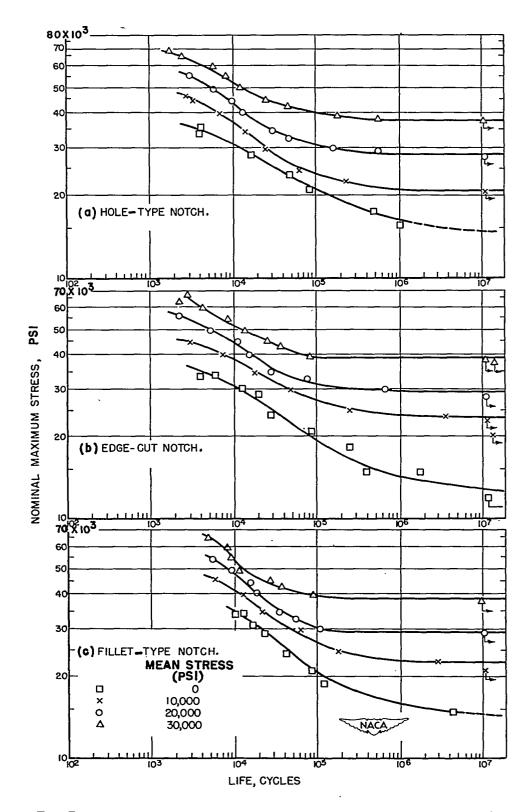


Figure 7.- Results of axial-load fatigue tests on notched 75S-T6 aluminum sheet specimens. $K_{\mbox{\scriptsize t}}$ = 2.0.

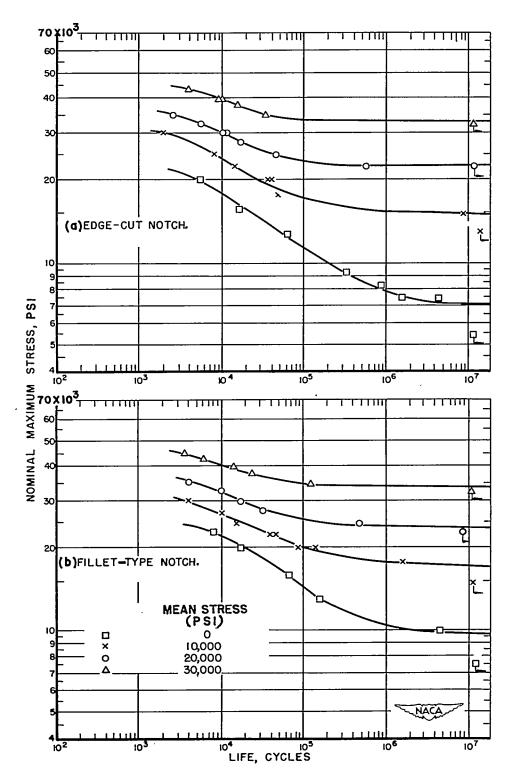


Figure 8.- Results of axial-load fatigue tests on notched 75S-T6 aluminum sheet specimens. $K_{\mbox{\scriptsize t}}=4.0$.

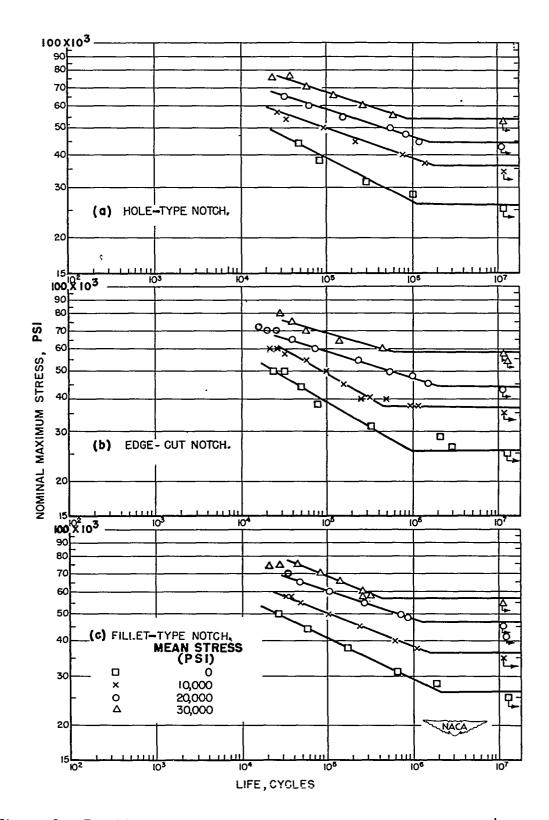


Figure 9.- Results of axial-load fatigue tests on notched SAE 4130 steel sheet specimens. $K_{\rm t}$ = 2.0.

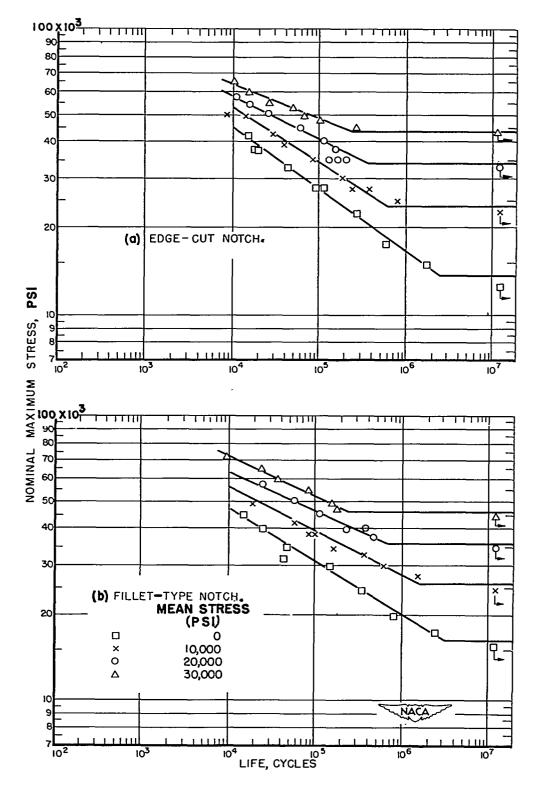


Figure 10.- Results of axial-load fatigue tests on notched SAE 4130 steel sheet specimens. $K_{\mbox{\scriptsize t}}$ = 4.0.

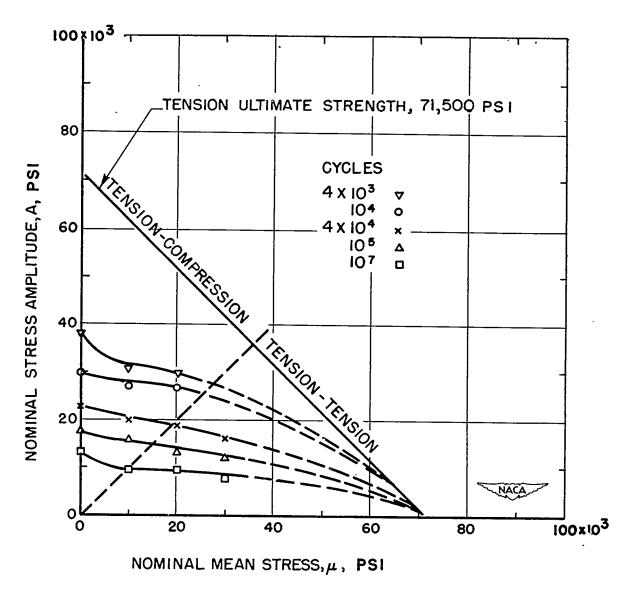


Figure 11.- Constant-lifetime curves for 24S-T3 aluminum sheet notched with a central hole. $K_{\mbox{\scriptsize t}}$ = 2.0.

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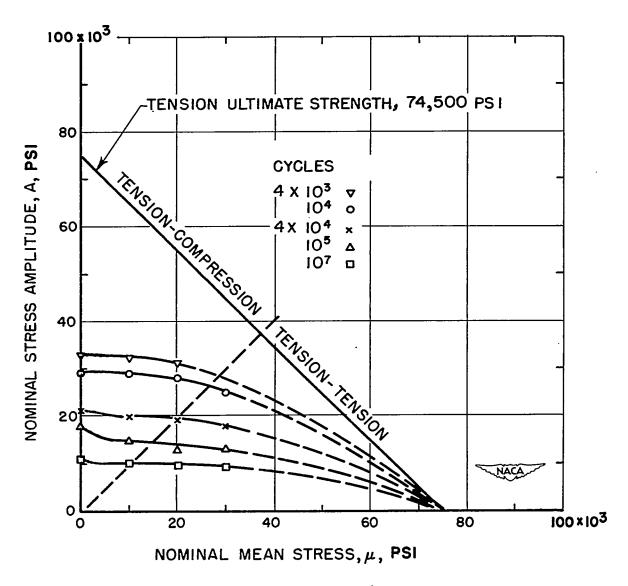


Figure 12.- Constant-lifetime curves for 24S-T3 aluminum sheet notched with symmetrical edge cuts. $K_{\mbox{\scriptsize t}}$ = 2.0.

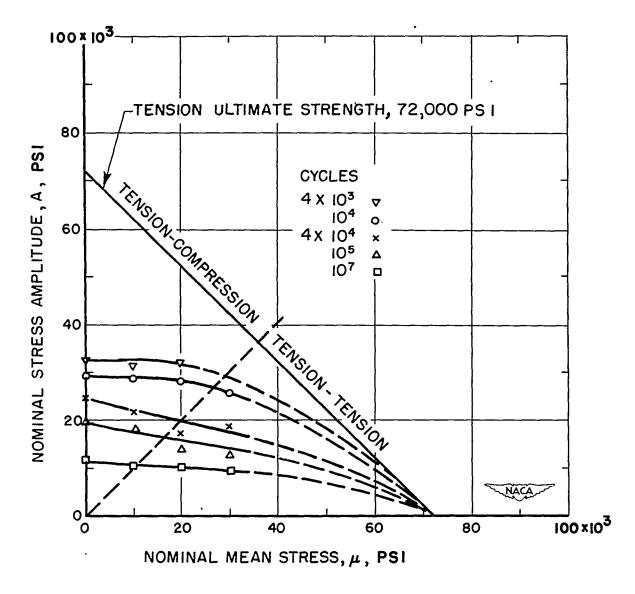


Figure 13.- Constant-lifetime curves for 24S-T3 aluminum sheet notched with symmetrical fillets. $K_{\mbox{\scriptsize t}}=2.0$.

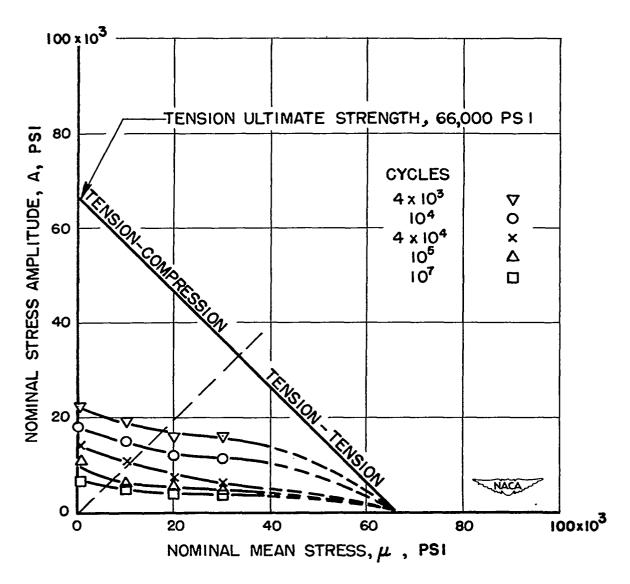


Figure 14.- Constant-lifetime curves for 24S-T3 aluminum sheet notched with symmetrical edge cuts. $K_{\mbox{\scriptsize t}}$ = 4.0.

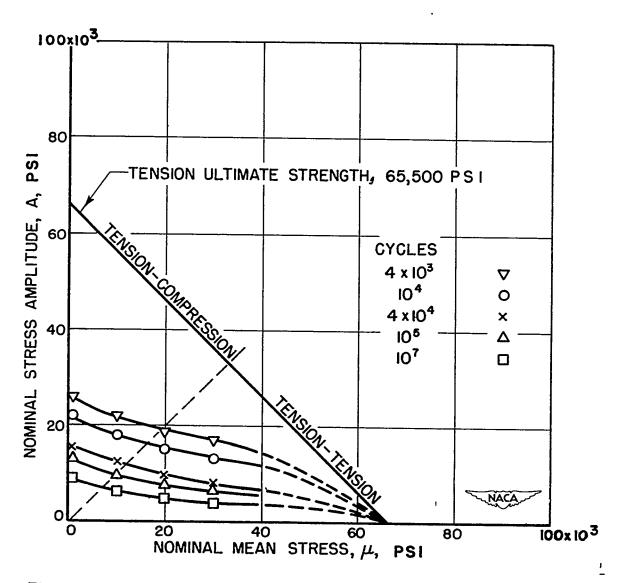


Figure 15.- Constant-lifetime curves for 24S-T3 aluminum sheet notched with symmetrical fillets. $K_{\mbox{\scriptsize t}}=4.0$.

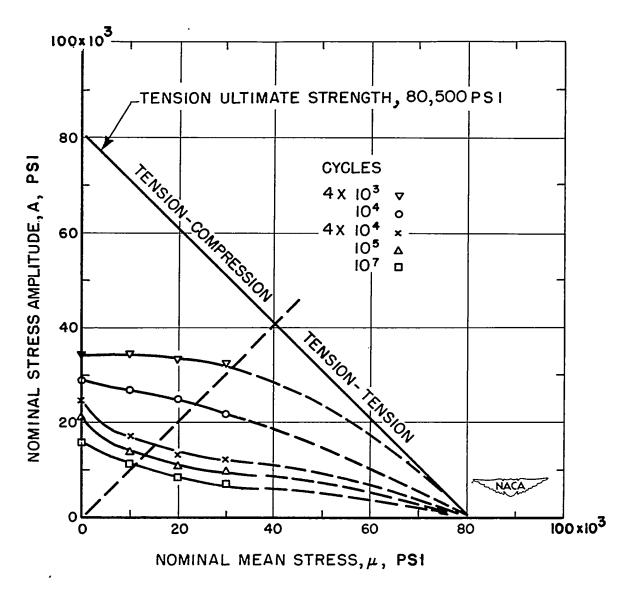


Figure 16.- Constant-lifetime curves for 758-T6 aluminum sheet notched with a central hole. $K_{\mbox{\scriptsize t}}$ = 2.0.

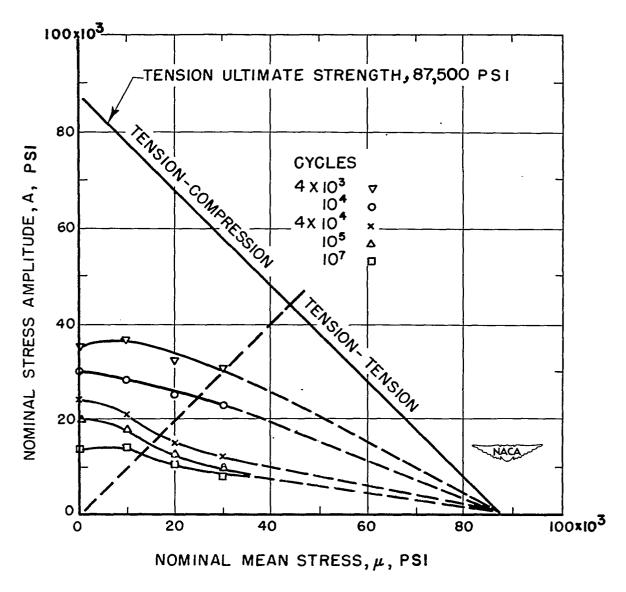


Figure 17.- Constant-lifetime curves for 75S-T6 aluminum notched with symmetrical edge cuts. $K_{\rm t}$ = 2.0.

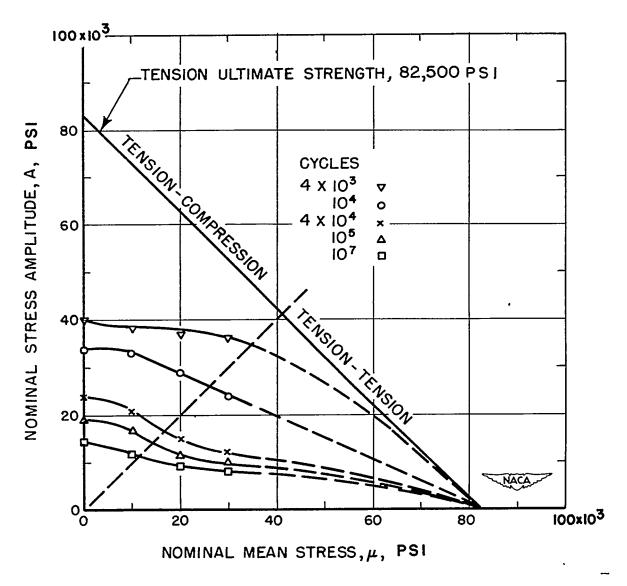


Figure 18.- Constant-lifetime curves for 758-T6 aluminum sheet notched with symmetrical fillets. $K_{\mbox{\scriptsize t}}$ = 2.0.

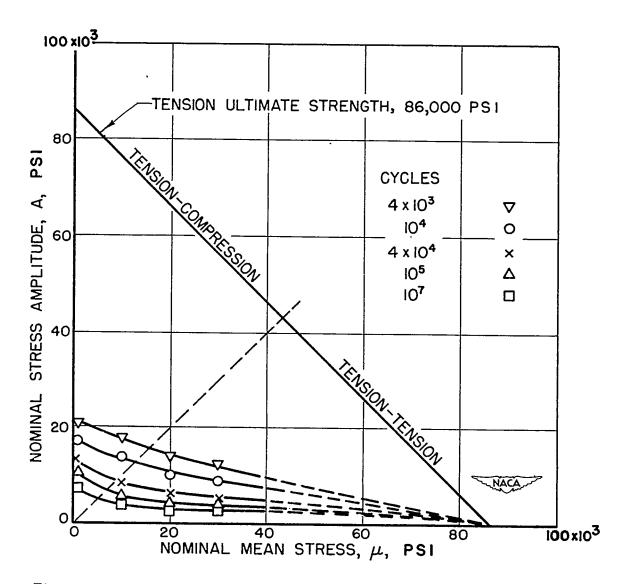


Figure 19.- Constant-lifetime curves for 75S-T6 aluminum sheet notched with symmetrical edge cuts. $K_{\mbox{\scriptsize t}}=4.0$.

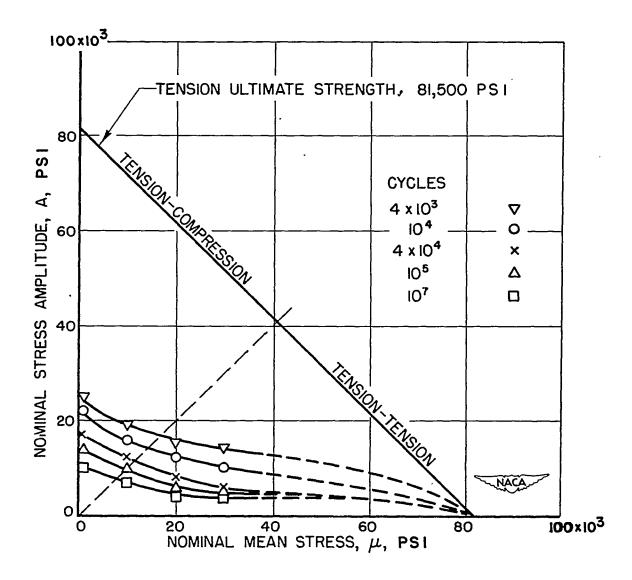


Figure 20.- Constant-lifetime curves for 75S-T6 aluminum sheet notched with symmetrical fillets. $\rm K_{t}$ = 4.0.

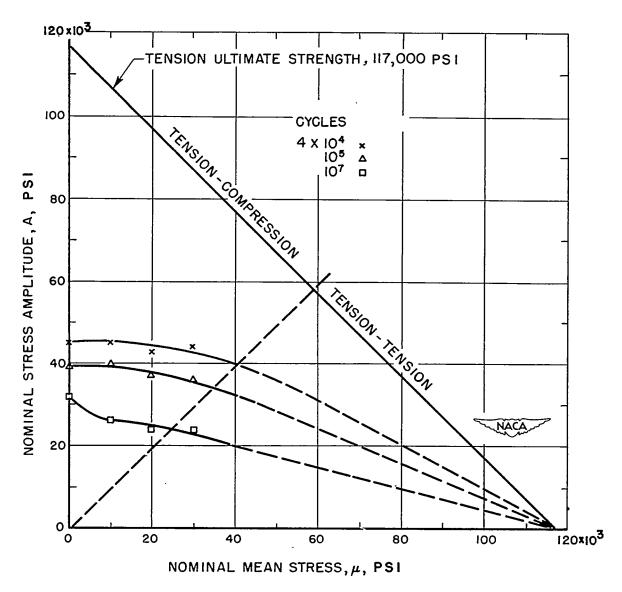


Figure 21.- Constant-lifetime curves for SAE 4130 steel sheet notched with a central hole. $K_{\mbox{\scriptsize t}}$ = 2.0.

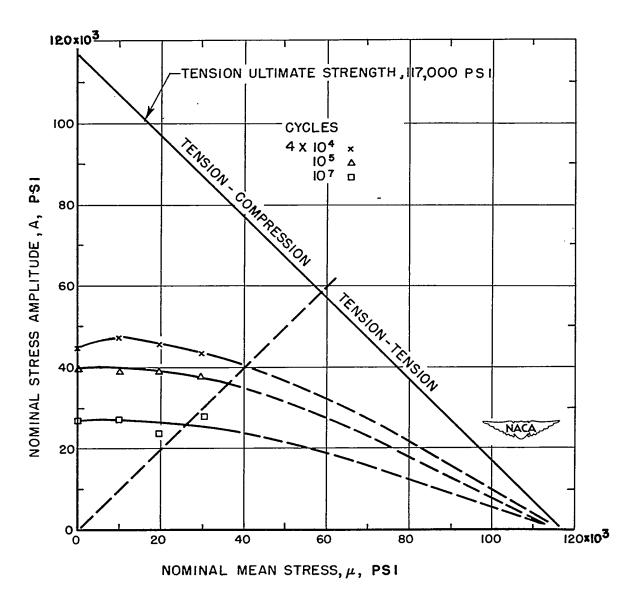


Figure 22.- Constant-lifetime curves for SAE 4130 steel sheet notched with symmetrical edge cuts. $K_t = 2.0$.

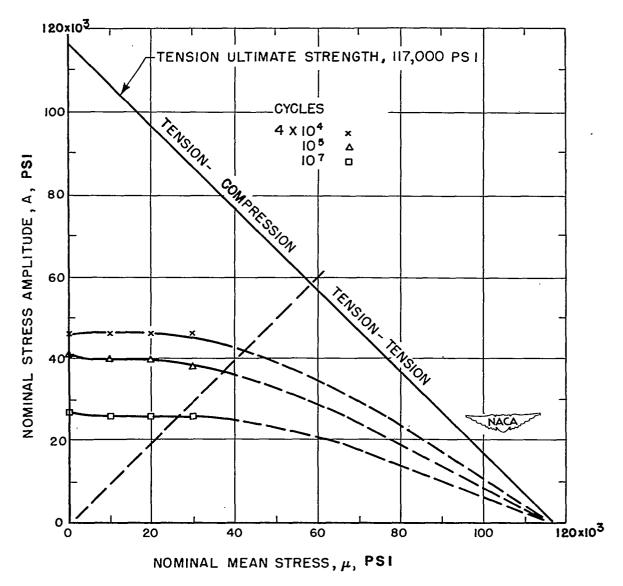


Figure 23.- Constant-lifetime curves for SAE 4130 steel sheet notched with symmetrical fillets. $K_{t}=2.0$.

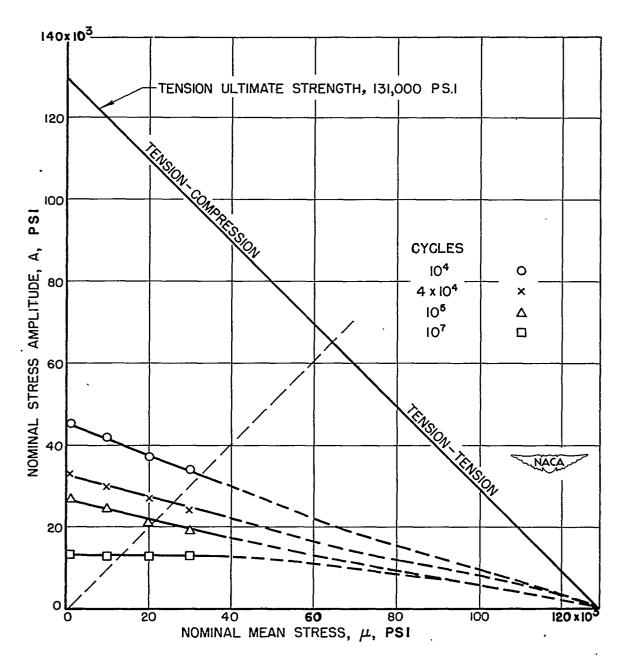


Figure 24.- Constant-lifetime curves for SAE 4130 steel sheet notched with symmetrical edge cuts. $K_{\rm t}$ = 4.0.

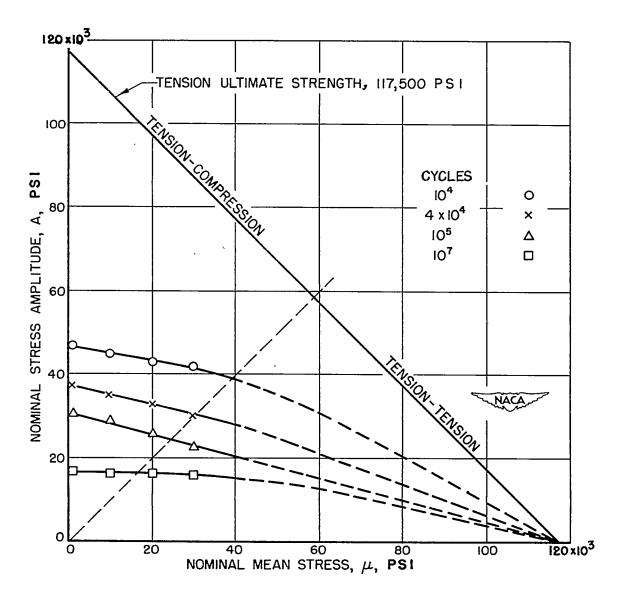


Figure 25.- Constant-lifetime curves for SAE 4130 steel sheet notched with symmetrical fillets. $K_{\rm t}$ = 4.0.

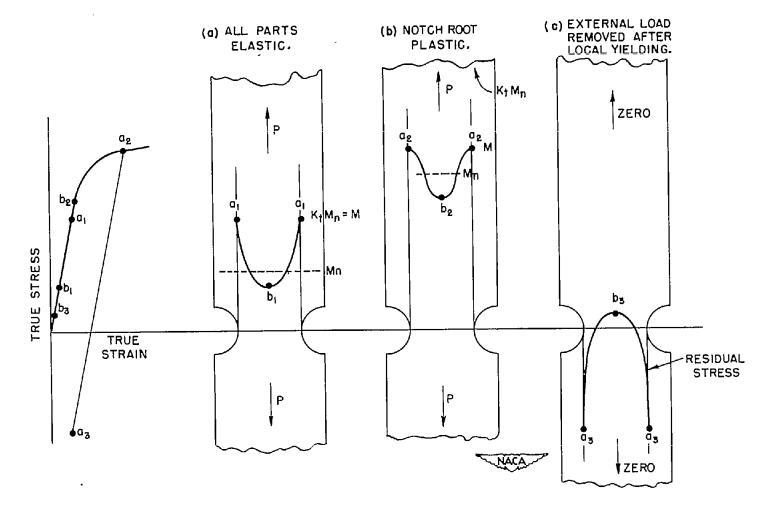


Figure 26.- Schematic representation of stress distribution in a notched specimen at various levels of applied stress. K_t, theoretical stress-concentration factor; M, peak stress; M_n, maximum nominal stress; a₁, a₂, a₃, b₁, b₂, b₃, stresses over cross section.

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